

CIVIL ENGINEERING-CE



GATE / PSUs

STUDY MATERIAL

STRENGTH OF MATERIALS





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CHAPTER

SIMPLE STRESSES AND STRAINS

STRESS (σ):

It is the internal resistance offered by a body against the deformation numerically, it is given as force per unit area.

Stress on elementary area ΔA ,

i.e.
$$\sigma = \lim_{\Delta A \rightarrow 0} \frac{\Delta F}{\Delta A} = \frac{dF}{dA} \quad (N / m^2)$$
 This unit is called

Pa(Pascal)

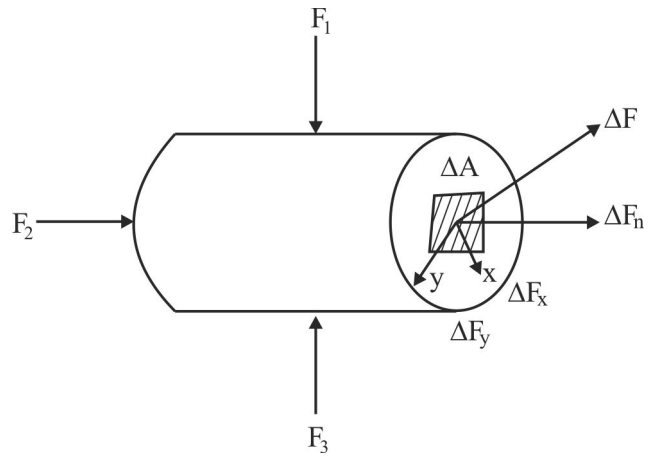
In case of normal stress dF always \perp (perpendicular) to area dA .

Pascal is a small unit in practice. These units are generally used

$$1 \text{ kPa} = 10^3 \text{ Pa} = 10^3 \text{ N/ m}^2$$

$$1 \text{ MPa} = 10^6 \text{ Pa} = 10^6 \text{ N/ m}^2$$

$$1 \text{ GPa} = 10^9 \text{ Pa} = 10^9 \text{ N/ m}^2$$



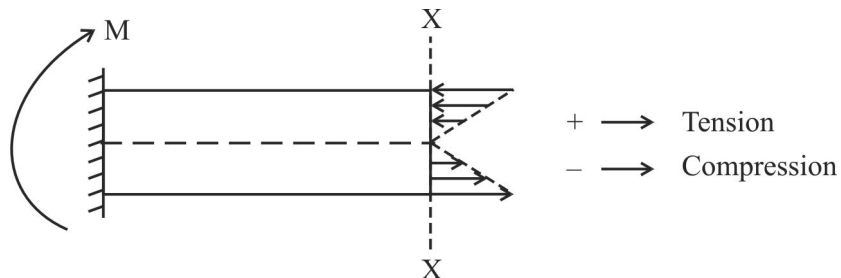
1. Normal Stress: It may be tensile or compressive depending upon the force acting on the material.

Tensile and compressive stresses are called **direct stresses**.

When, $\sigma > 0$, Tensile

When, $\sigma < 0$, Compressive

➤ The other types of normal stress is bending normal stress.



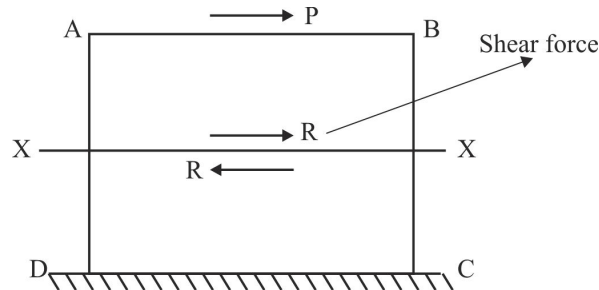
Bending stress are linearly distributed from zero at neutral axis to maximum at surface.

➤ In bending, the cross-sectional area rotates about transverse axis and the axis about which the cross-sectional area rotates is called neutral axis hence in bending, neutral axis is always transverse axis.

2. **Shear Stress (τ):** It is the intensity of shear resistance along a surface (Let X-X).

$$\tau = \frac{\text{Shear force}}{\text{Shear Area}} \text{ (N/m}^2\text{)}$$

In case of shear stress force always parallel to the sheared area *i.e.* P is parallel to sheared area in figure.



3. **Conventional or Engineering Stress (σ_0):** It is defined as the ratio of load (P) to the original area of cross-section (A_0):

$$\therefore \sigma_0 = \frac{P}{A_0}$$

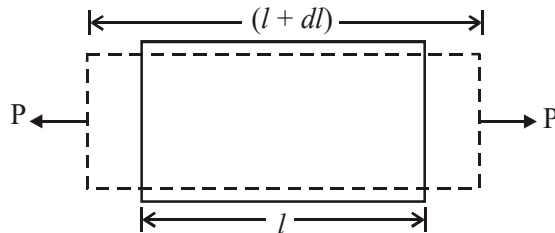
4. **True Stress (σ):** It is defined as the ratio of load (P) to the instantaneous area of cross-section (A):

$$\therefore \sigma = \frac{P}{A} \text{ or, } \sigma = \sigma_0(1 + \epsilon) \text{ Where } \epsilon = \text{strain } \left[\begin{matrix} Al = A_0l_0 \\ l = l_0(1 + \epsilon) \end{matrix} \right] \text{ Initial volume = Final volume}$$

STRAINS (ϵ):

It is defined as the change in length per unit length. It is a dimensionless quantity.

$$i.e. \epsilon = \frac{\text{change in length}}{\text{original length}} = \frac{dl}{l}$$



1. **Conventional or Engineering strain:** It is defined as the change in length per unit original length.

$$\epsilon = \frac{l - l_0}{l_0}$$

Where,

l = Deformed length

l_0 = Original length

e.g. from above figure.

$$\varepsilon = \frac{l + dl - l}{l} \quad \boxed{\varepsilon = \frac{dl}{l}}$$

2. **Natural Strain:** It is defined as the change in length per unit instantaneous length.

$$\bar{\varepsilon} = \int_{l_0}^l \frac{dl}{l} = \ln \frac{l}{l_0} = \ln \left(\frac{A_0}{A} \right) = 2 \ln \left(\frac{d_0}{d} \right)$$

Also, $\therefore \bar{\varepsilon} = \ln(1 + \varepsilon)$

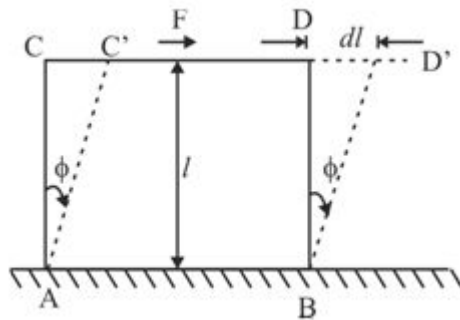
$$\Rightarrow 1 + \varepsilon = e^{\bar{\varepsilon}}$$

$$\Rightarrow \varepsilon = e^{\bar{\varepsilon}} - 1$$

Volume of the specimen is assumed to be constant during plastic deformation

$\therefore \boxed{A_0 L_0 = AL}$ -Valid till neck formation.

3. **Shear Strain (ϕ):** It is the strain produced under the action of shear stresses.



Shear Strain = $\tan \phi$

For small strain, $\boxed{\tan \phi \approx \phi}$

From figure, $\Delta ACC'$ or $\Delta BDD'$

$$\tan \phi = \frac{dl}{l} = \frac{CC'}{l}$$

$$\boxed{\phi = \frac{dl}{l} = \frac{\text{Transverse displacement}}{\text{Distance from lower face}}}$$

➤ Shear strain cause deformation in shape but volume remains same.

4. **Superficial strain (ε_s):** It is defined as the change in area of cross section per unit original area.

$$\boxed{\varepsilon_s = \frac{A - A_0}{A_0}}$$

Where, A = Final area A_0 = Original area

5. **Volumetric Strain (ϵ_v):** It is defined as the change in volume per unit original volume.

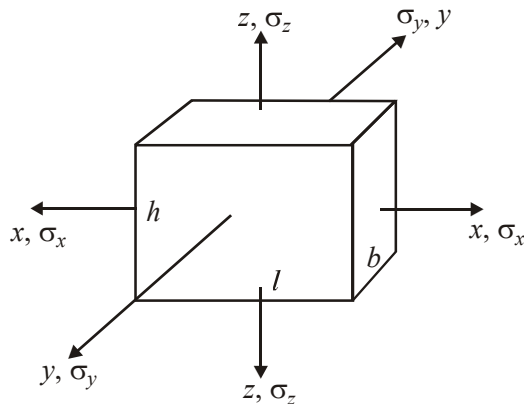
$$\epsilon_V = \frac{V - V_0}{V_0}$$

Where, V = Final volume V_0 = Original volume

➤ Stress and strain are tensor (*neither vector nor scalar*) of 2nd order.

$$\text{Volumetric strain } \epsilon_V = \epsilon_x + \epsilon_y + \epsilon_z$$

Volumetric strain for various shapes:



(i) **Rectangular body:**

$V = lbh$ on partial differentiation

$$\delta V = \delta l(b.h) + \delta b(l.h) + \delta h(b.l)$$

$$\epsilon_V = \frac{\delta V}{V} = \frac{\delta l}{l} + \frac{\delta b}{b} + \frac{\delta h}{h}$$

$$\epsilon_V = \epsilon_x + \epsilon_y + \epsilon_z$$

Note: $\epsilon_x, \epsilon_y, \epsilon_z$ are the strain corresponding to the stresses $\sigma_x, \sigma_y, \sigma_z$ in x -direction, y -direction, z -direction respectively

$$\epsilon_V = \frac{\sigma_x + \sigma_y + \sigma_z}{E} (1 - 2\nu)$$

μ —Poisson Ratio

$\mu = 0.5$ For rubber

(ii) **For cylindrical body:**

$$V = \frac{\pi}{4} d^2 l$$

$$\delta V = 2 dl \cdot \delta d \cdot \frac{\pi}{4} + \frac{\pi}{4} d^2 \delta l$$

$$\epsilon_v = \frac{\delta V}{V} = 2 \frac{\delta d}{d} + \frac{\delta l}{l}$$

$$\epsilon_v = 2\epsilon_d + \epsilon_l$$

(iii) For spherical body

$$\epsilon_v = 3 \frac{\delta d}{d}$$

$$V = \frac{4}{3} \pi r^3$$

$$d = 2r$$

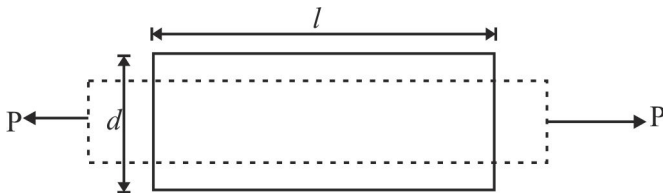
Gauge Length: It is that portion of the test specimen over which extension or deformation is measured.

i.e. this length is used in calculating strain value.

Poisson's ratio (μ or $\frac{1}{m}$): Value of μ varies between (-1 to 0.5)

The ratio of the lateral strain to longitudinal strain is called the Poisson's ratio.

$$\mu = \frac{\text{Lateral strain}}{\text{Longitudinal strain}} \quad \text{or} \quad \mu = \frac{-\left(\frac{\delta d}{d}\right)}{\left(\frac{\delta l}{l}\right)}$$



- For a given material, the value of 'μ' is constant throughout the linearly elastic range.
- For most of the metals the value of 'μ' lie between 0.25 – 0.42
- 'μ' varies from (-1 to 0.5)

Note: 'μ' for ductile material is greater than 'μ' for brittle metals.

Table

Material	Value of 'μ'	Remarks
Cork	0	∴ Used in bottle to withstand pressure
Foam	-1	
Rubber	0.5	
Concrete	0.1 – 0.2	
C.I.	0.23 – 0.27	

For cork μ = 0

For rubber μ = 0.5

For concrete μ = 0.1 – 0.2

Isotropic Material: These materials have same elastic properties in all directions.

No. of independent elastic constants = 2, *i.e.* if any of 2 elastic constants is known then other can be derived.

Orthotropic Material: The number of independent elastic constants is 9.

Anisotropic materials: These materials don't have same elastic properties in all directions.

Elastic moduli will vary with additional stresses appearing. ∴ There is a coupling between shear stress and normal stress for an isotropic material.

The number of independent elastic constants is 21.

Hooke's Law: It states that when a material is loaded such that the intensity of stress is within a certain limit, the ratio of the intensity of stress to the corresponding strain is a constant which is characteristics of that material.

Note: The certain limit is called proportionality limit but for most practical purposes, proportionality limit equals elastic limit.

i.e. $\frac{\text{Stress}}{\text{Strain}} = \text{Constant} \neq E$ *i.e.,* $\sigma = E \epsilon$

Where, $E = \text{Young's Modulus (N/m}^2\text{)}$

Or

Modulus of Elasticity

- For steel, value of $E = 210 \text{ GPa}$ ($1 \text{ GPa} = 10^3 \text{ N/m}^2$)
- For aluminum, value of $E = 73 \text{ GPa}$ $E_{Al} \approx \frac{1}{3} \text{rd } E_{\text{steel}}$
- For Plastic, value of $E = 1 \text{ GPa} - 14 \text{ GPa}$

Note : As flexibility increases, value of young's modulus decreases.

It is resistance to elastic strain.

Shear Modulus of Elasticity OR Modulus of Rigidity (G or C): It is defined as the ratio of shearing stress to shearing strain.

$$G \text{ or } C = \frac{\text{Shear stress}}{\text{Shear strain}} \text{ i.e. } \tau \propto \phi$$

Bulk Modulus (K):

It is defined as the ratio of uniform stress intensity to volumetric strain within the elastic limits.

$$K = \frac{\text{Normal Stress}}{\text{Volumetric Strain}}$$

Note: Elastic constant relationship

- (i) $E = 2C(1 - \mu)$, where, ν = Poisson's ratio.
- (ii) $E = 3K(1 - 2\mu)$
- (iii) $\mu = \frac{3K - 2C}{6K + 2C}$
- (iv) $E = \frac{9KC}{3K + C}$

STRESS-STRAIN DIAGRAM:

I. Ductile material (Mild Steel):

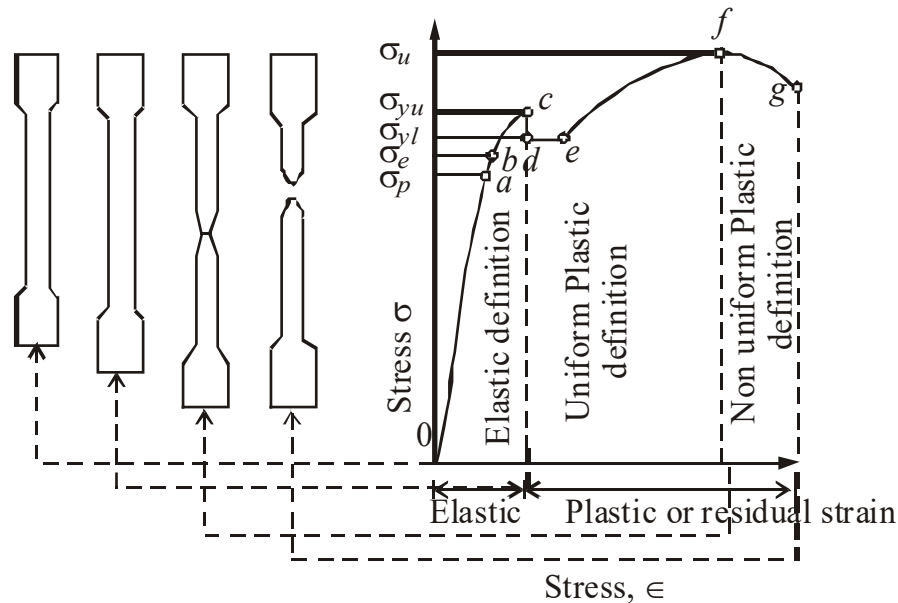


Figure: Typical stress-strain diagram for a ductile material

- Point 'a' → Limit of proportionality: Up to this point 'a', Hooke's law is obeyed; 'oa' is a straight line. Stress corresponding to this point is called 'proportional limit stress, σ_p '

Comparison of Engineering and true stress strain curve:

- The true stress-strain curve is also known as **flow curve**.
- True stress-strain curve gives a true indication of deformation characteristics because it is **based on the instantaneous dimension of specimen**.
- In engineering stress-strain curve, the stress drops down after necking since it is **based on the original area**.
- In true stress strain curve, the stress however increases after necking since the cross section area of the specimen **decreases rapidly after necking**.
- The flow curve of many metals in the region of uniform plastic deformation can be expressed by **simple power law**.

$$\sigma_T = K(\epsilon_T)^n$$

where, K is the strength co-efficient, σ_T is time stress.

n is the strain hardening coefficient.

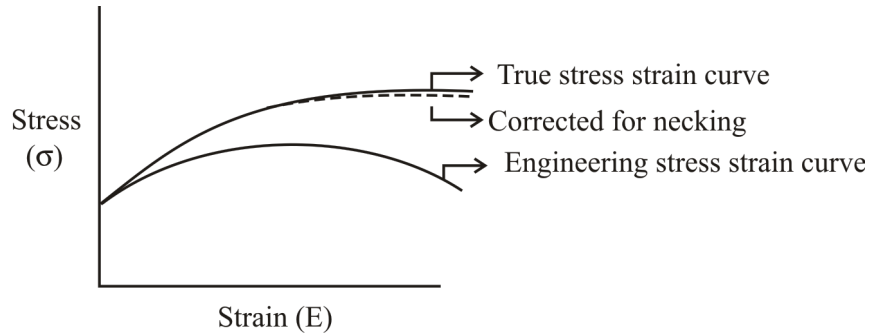
n = 0 for perfectly plastic solid

$n = 1$ In elastic solid

For most metals $0.1 < n < 0.5$

$\sigma_{True} > \sigma_{Nominal} \rightarrow$ if force is tensile, since area decreases.

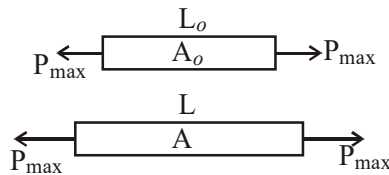
$\sigma_{True} > \sigma_{Nominal} \rightarrow$ if force is compressive, since area increase.



➤ Relation between ultimate tensile strength and true stress at maximum load.

Ultimate tensile strength $\sigma_u = \frac{P_{max}}{A_o}$

True stress at maximum load = $(\sigma_u)_T = \frac{P_{max}}{A}$



True strain at max load $(\epsilon_T) = \ln \frac{A_o}{A}$ or $\frac{A_o}{A} = e^{\epsilon_T}$

Eliminating P_{max} we get

$$(\sigma_u)_T = \frac{P_{max}}{A} \times \frac{A_o}{A_o}$$

$$= \frac{P_{max}}{A_o} \times e^{\epsilon_T}$$

$$\Rightarrow (\sigma_u)_T = \sigma_u e^{\epsilon_T}$$

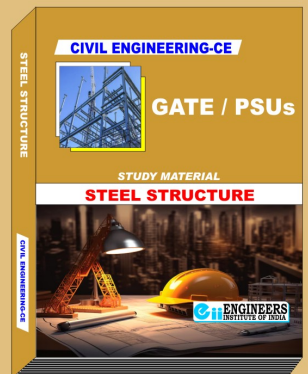
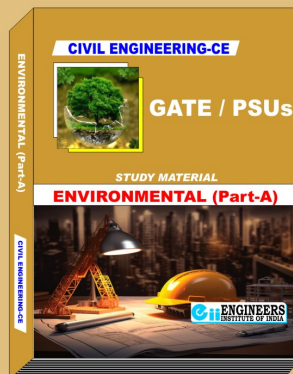
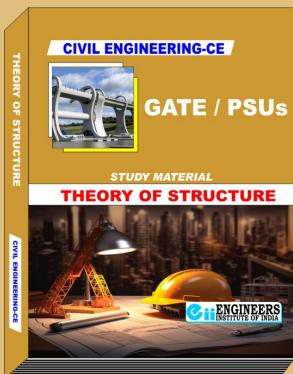
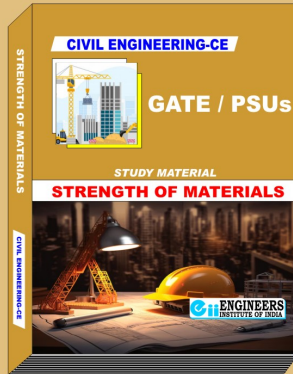
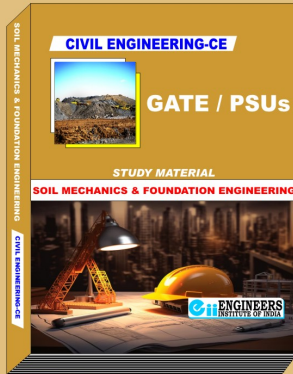
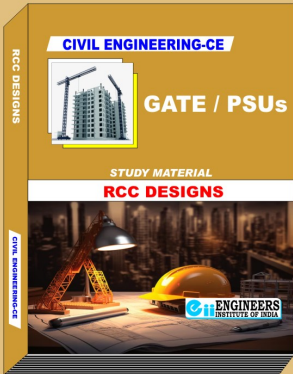
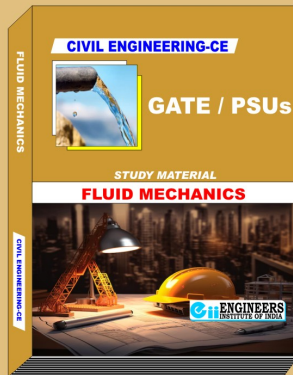
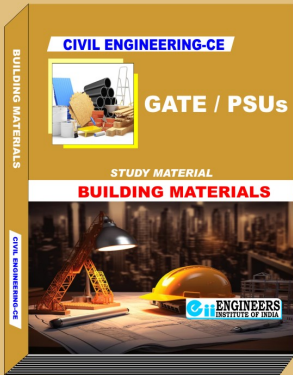
Here, P_{max} is the max force.

A_o = original cross section area

A = instantaneous cross section area

- Based on the above theory two examples has been provided.

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